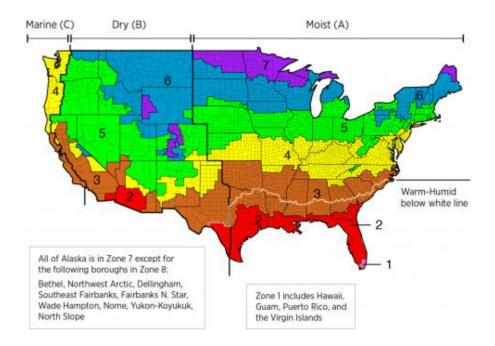
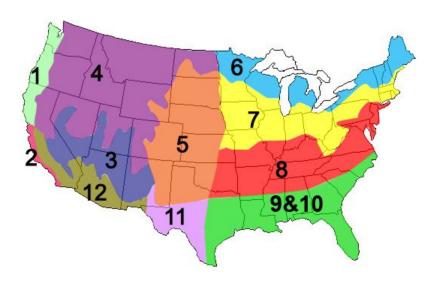
Dehumidification & Ventilation

David Treleven
Ultra-Aire
2/27/19

Green Grass Climates

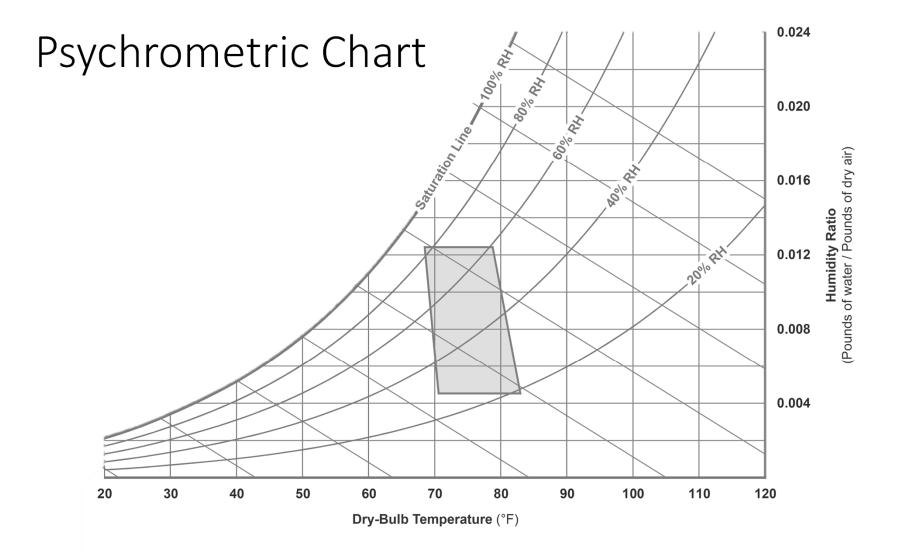


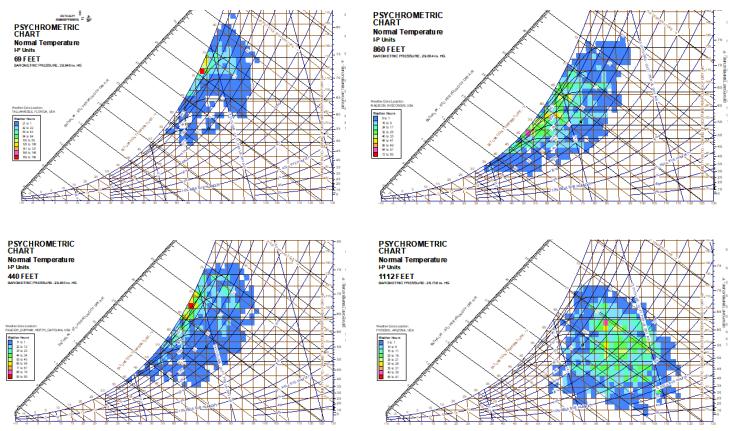


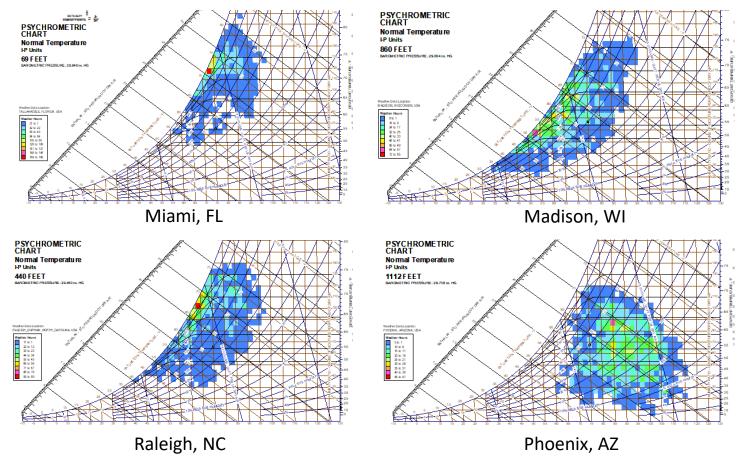
Green Grass Climates

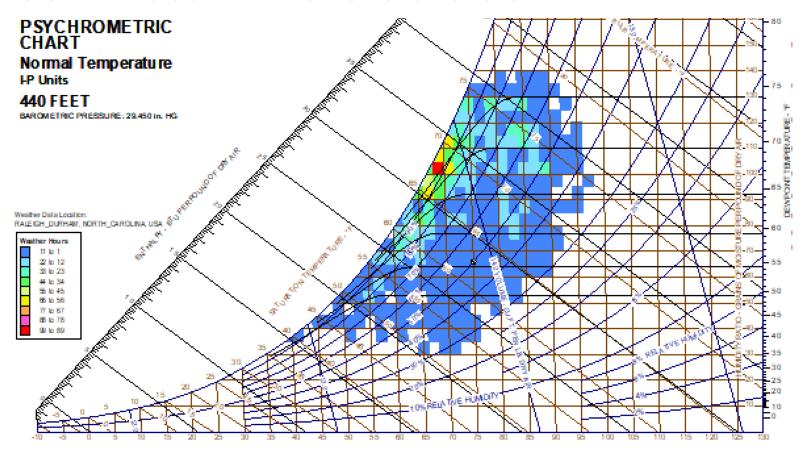


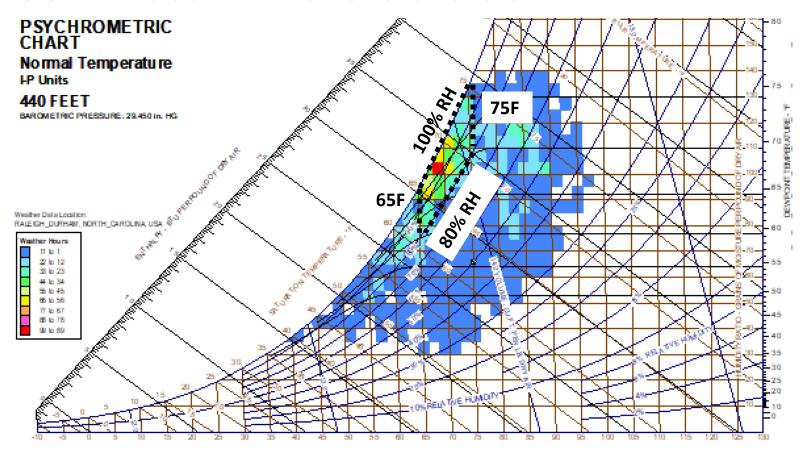


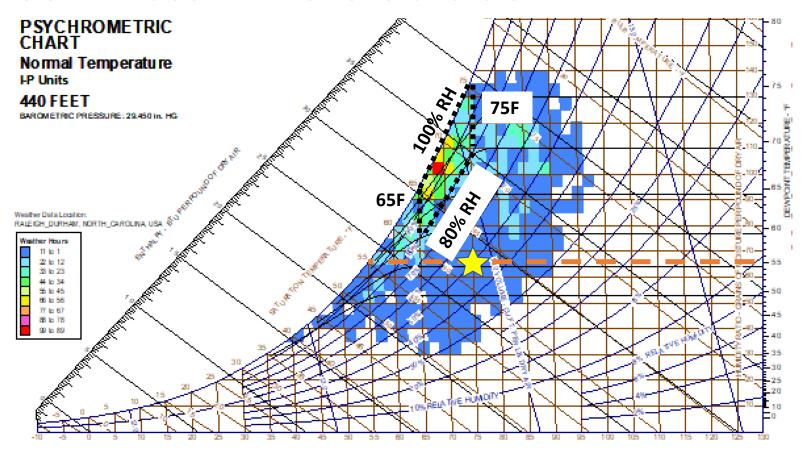




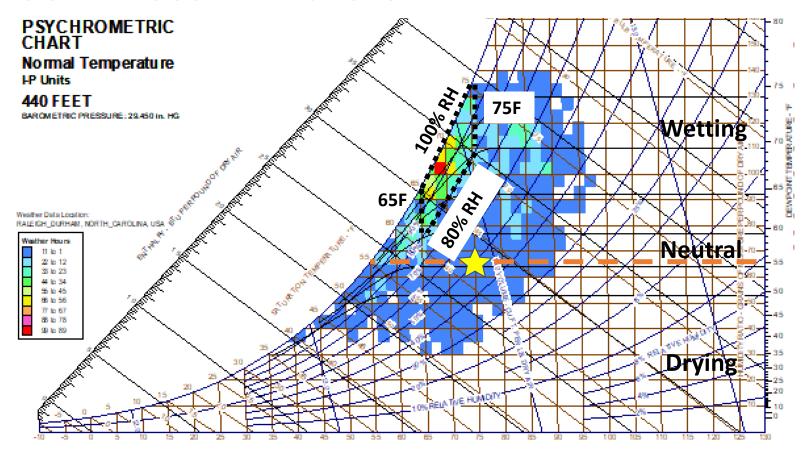








Green Grass Climate OA



Ventilation Load

Conditions	85F/60%		76F/95%	
Vent Rate	50CFM	100CFM	50CFM	100CFM
Sensible	540 btuh	1080 btuh	54 btuh	108 btuh
Latent	1563 btuh	3128 btuh	2261 btuh	4523 btuh
Total	2103 btuh	4208 btuh	2315 btuh	4631 btuh

^{*} Interior Conditions 75F 50%RH

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Dehumidification and Cooling Loads From Ventilation Air

By Lewis G. Harriman III

Member ASHRAE

Dean Plager and Douglas Kosar

Member ASHRAE

inety-five years since Willis Carrier began the modern era of air conditioning by dehumidifying a printing plant, our industry is becoming more concerned with the importance of controlling humidity in buildings. In part, this concern stems from indoor air quality problems associated with excess moisture in air-conditioning systems. But more universally, the need for ventilation air has forced HVAC equipment (originally optimized for high efficiency in removing sensible heat loads) to remove high moisture loads.¹

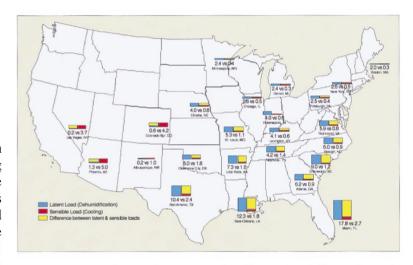
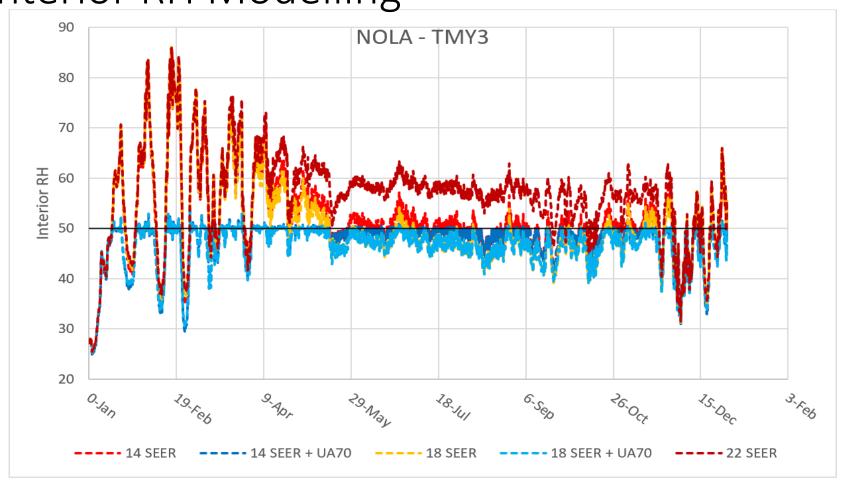
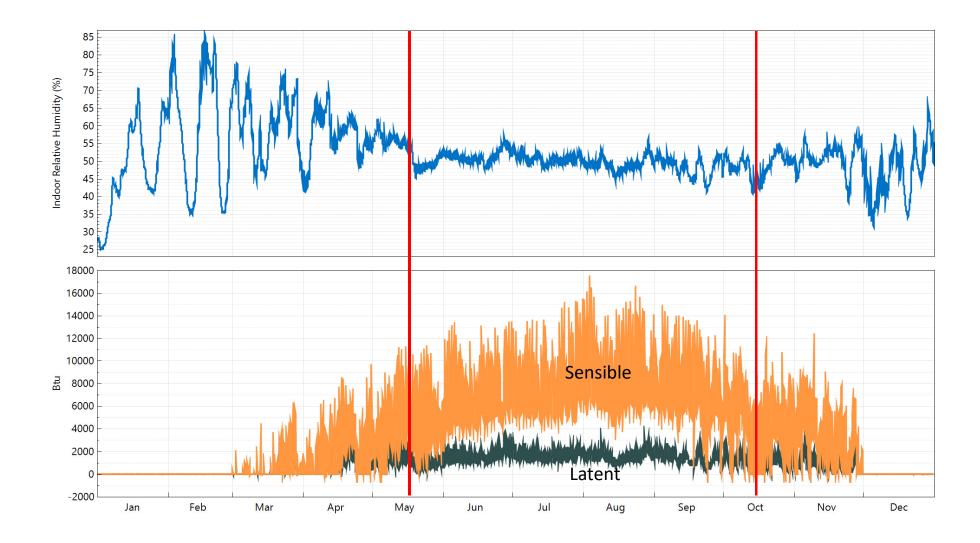


Fig. 1: Map of Ventilation Load Indexes (VLI) for selected locations.

Interior RH Modelling





Variable Capacity Equipment

- Higher AVG Supply Air Temp (~60F typical 55F)
- Less latent capacity when turned down
 - SHR 0.65
 - 3000 btuh Total ~2000 btuh Sensible & 1000 btuh Latent



Variable Capacity Equipment

- Higher AVG Supply Air Temp (~60F typical 55F)
- Less latent capacity when turned down

Outcide Air

- SHR 0.65
- 3000 btuh Total ~2000 btuh Sensible & 1000 btuh Latent

Ou	itside Air			
	Conditions	tions 76F/95%		
	Vent Rate	50CFM	100CFM	
	Sensible	54 btuh	108 <u>btuh</u>	
	Latent	2261 <u>btuh</u>	4523 <u>btuh</u>	
	Total	2315 htub	4631 htub	



Increased Insulation/Air-Tightness

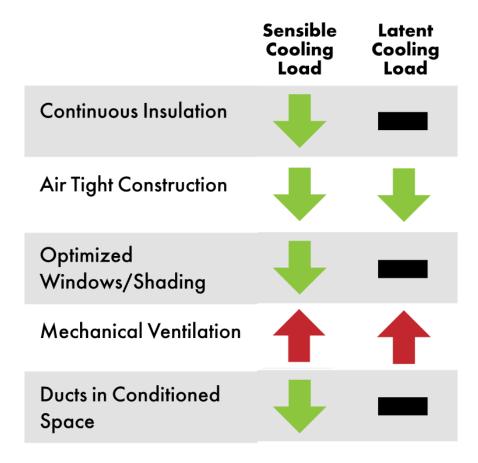




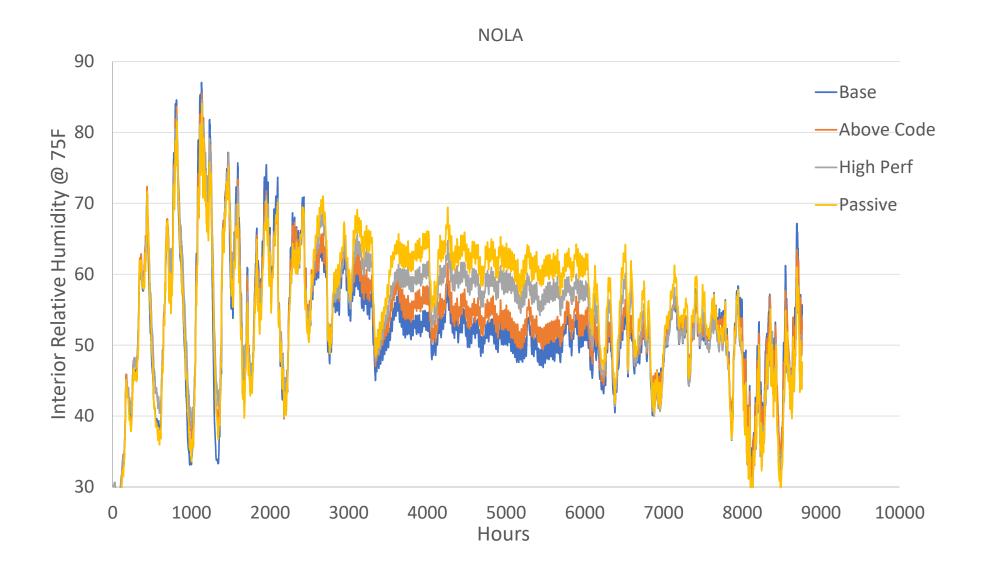




Low-Sensible Load Houses



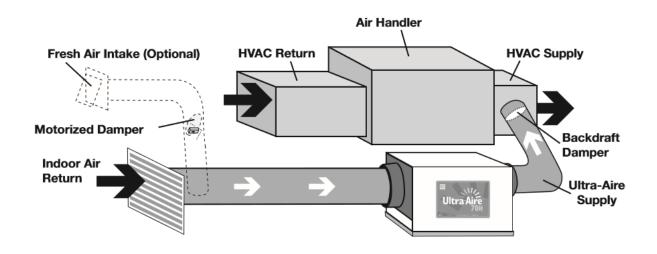




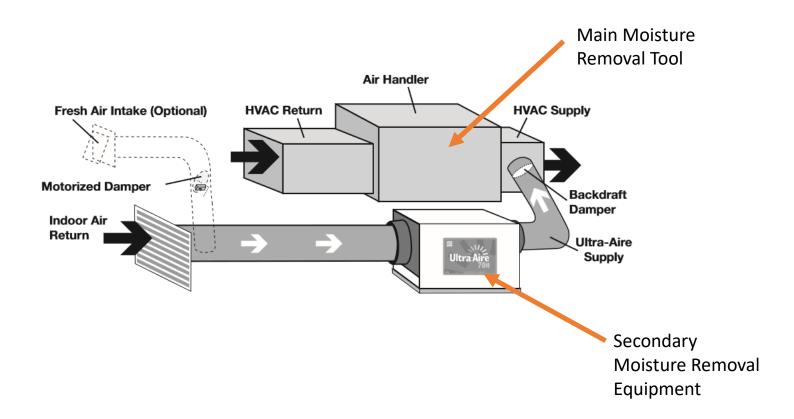
Ventilating/Whole Home Dehumidifiers



Ventilating Dehumidifer – Supply Ventilation



Ventilating Dehumidifer



Dehumidifier Performance

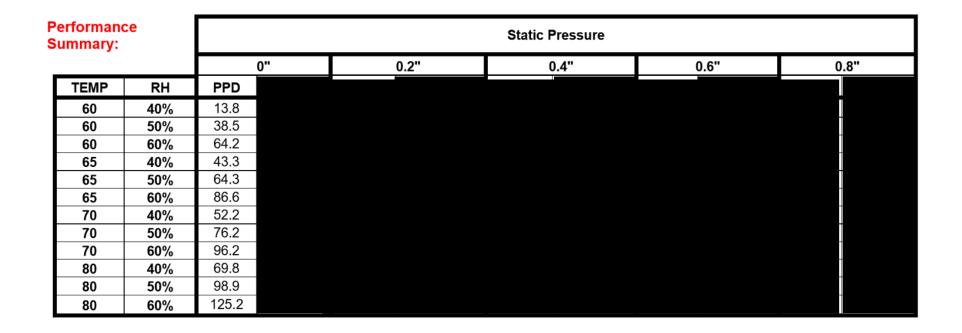
Important Dehumidifier Information:

- Dehumidifiers are rated @ 80F/60%RH 0.0 IWG (AHAM)
 - New 2019 DOE 73F/60%RH 0.2 IWG
- Pint ~ 1000 Btu/h
- Heat generated 3.46 Btu/h per Watt & 1000 Btu/h per pint of water removed

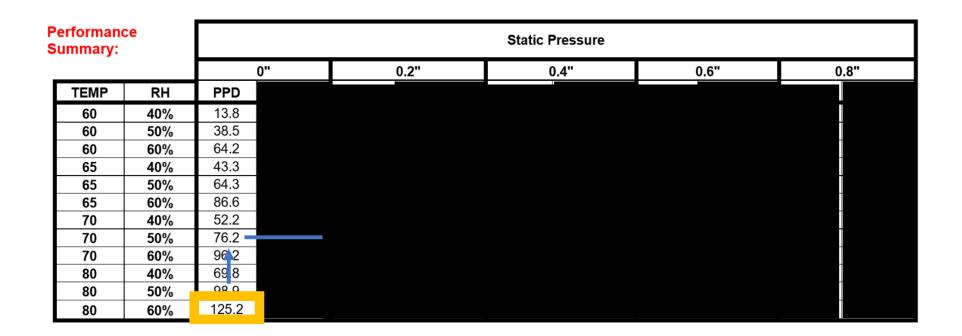


@ 75F/50%RH – 0.2 IWG Latent – 1700 Btu/h Sensible added - 3500 Btu/h

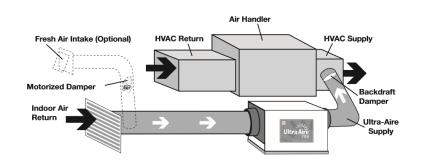
Dehumidifier Performance

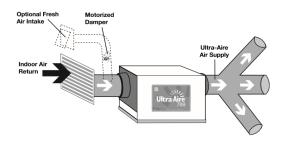


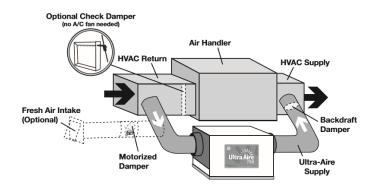
Dehumidifier Performance

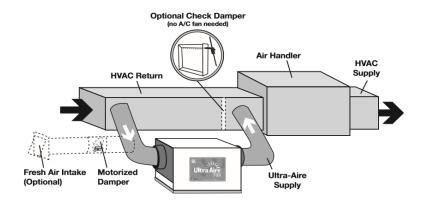


Whole Home Dehumidifier Installation

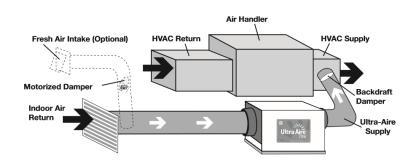


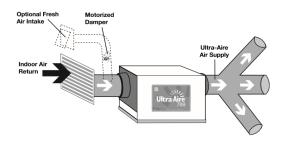


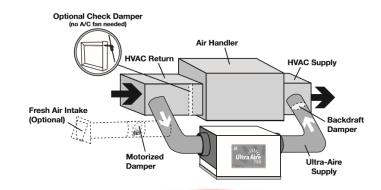


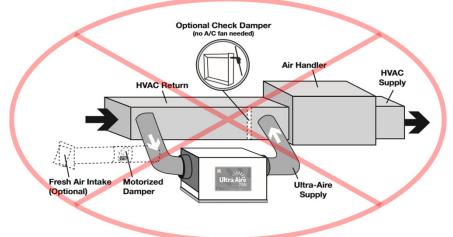


Whole Home Dehumidifier Installation





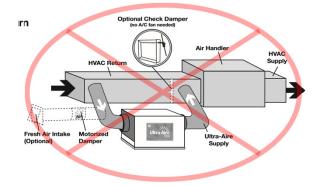




FSEC – June 2018 – FSEC-CR-2028-18

DHU ducted from/to central return had the highest daily energy use and resulted in two primary causes of latent performance degradation.

- DHU air degraded central latent cooling performance during simultaneous operations of both AC and DHU appliances. Temporary steady-state testing, with both the central cooling system and DHU operating at the same time, found that the central cooling latent performance was decreased by 28% compared to when no DH was operating at the same time.
- DHU air re-evaporated water off of warm central coil when AC was cycled off.
 Temporary steady-state testing just after the central system cycled off, with the DHU operating 28 continuous minutes after, measured a total 1.5 lbs of water re-evaporated off of the central cooling coil (rate of 3.2 lb/h back into condition space).
- O During one 15 minute period observation of uninterrupted monitoring, the moisture pulled out of the room air by the DHU was at about the same rate that was being reevaporated off of the central cooling coil while the AC was cycled off. In this instance the DHU coil rate of latent removal was -1.8 lb/h and the latent heat due to evaporation from the central cooling coil was +1.9 lb/h into the space while the DH was operating steady and the central cooling system had remained naturally cycled off 1.25 hours prior during very low cooling load period in the early morning.



100% Outside Air?

OA Conditions 85F/60%/DP 70

Vent Rate 150CFM

Interior Conditions 75F/50%/DP 55F



50 CFM OA 100 CFM IA



100% Outside Air?

OA Conditions 85F/60%/DP 70

Vent Rate 150CFM

Interior Conditions 75F/50%/DP 55F

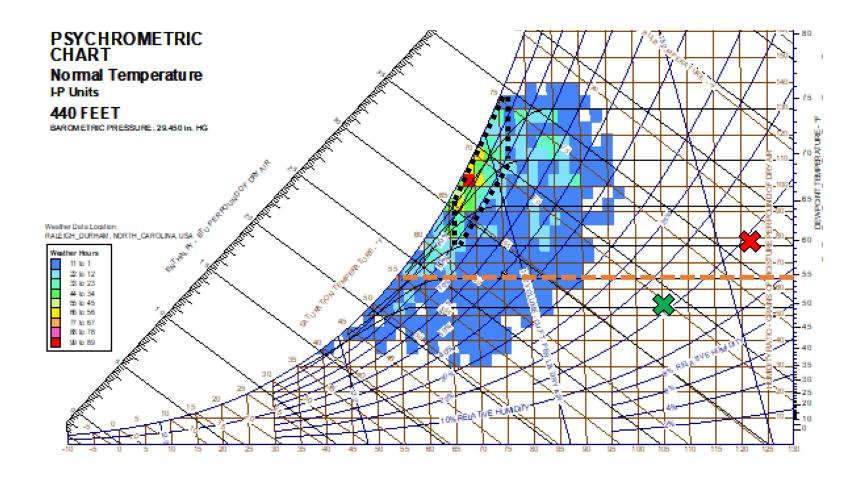


50 CFM OA 100 CFM IA

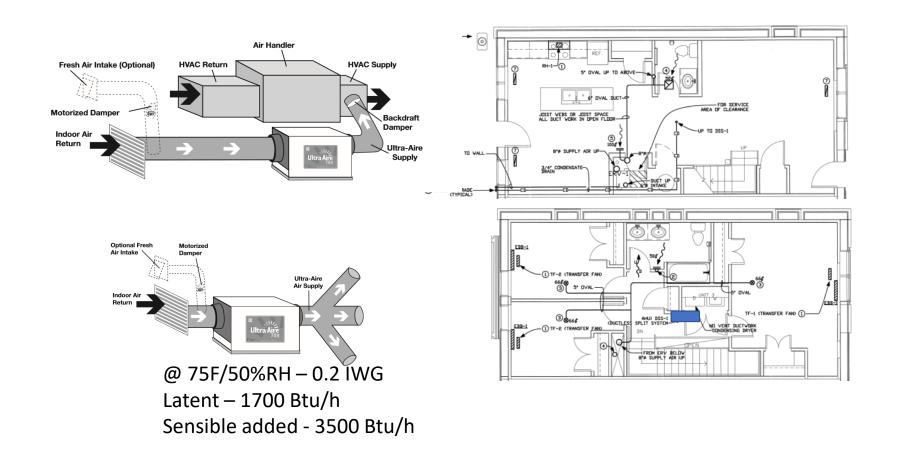


LAT ~ 122F/14%/DP 60F

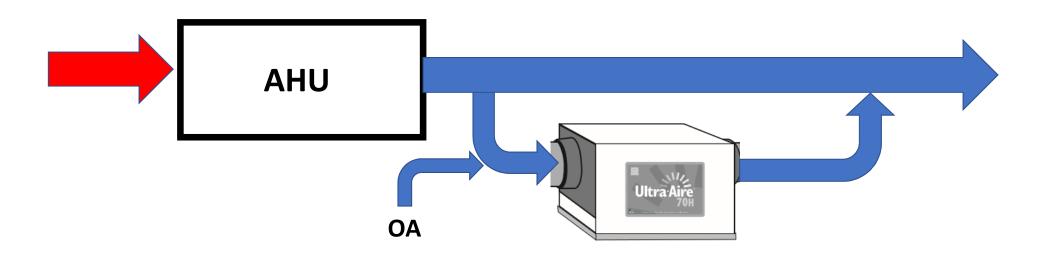
LAT ~ 105F/16%/DP 50F



Whole Home Dehumidifier Installation



Whole Home Dehumidifier Installation w/ Ducted Mini-Split



Whole Home Dehumidifier Installation w/ **Ducted Mini-Split** Exiting Air Dry & Close to Room Temp **Entering Air Closer to Dew Point AHU** AHU fan ON & cooling higher % of time Higher PPD @ OA **Lower Power** Operational

Static 0.0"

ERVs in Humid Climates

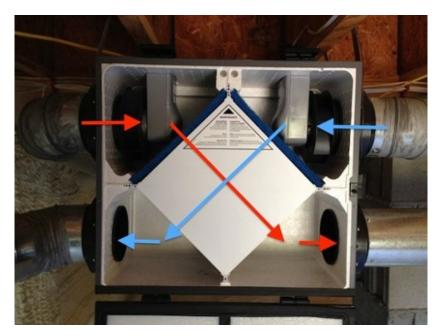
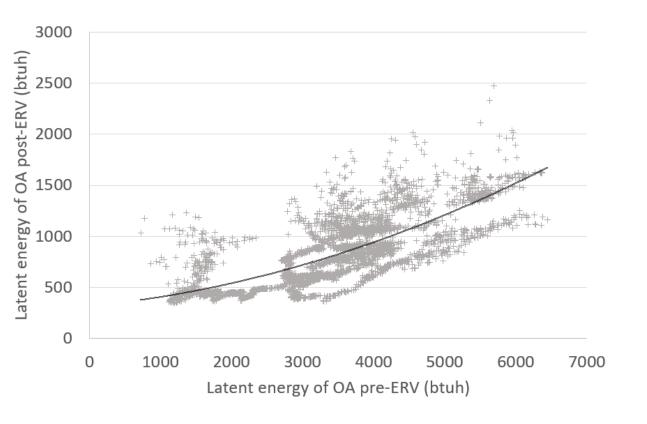
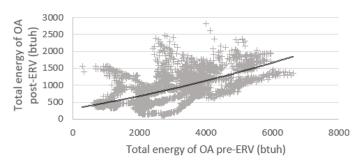
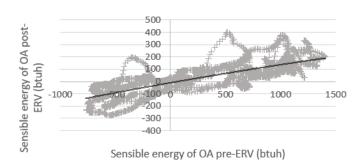
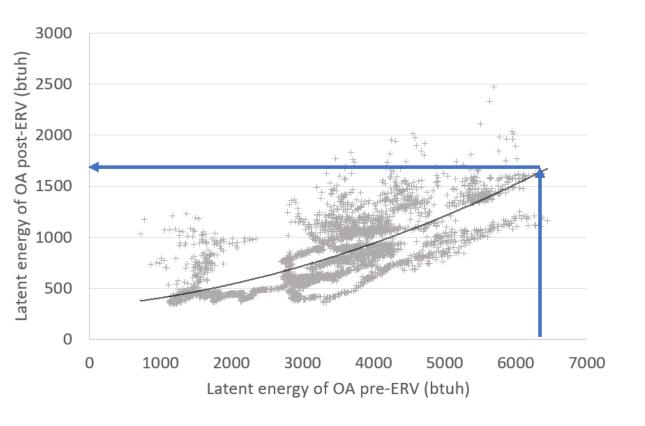


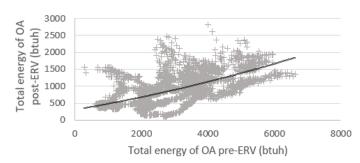
Photo Credit: Energy Vanguard

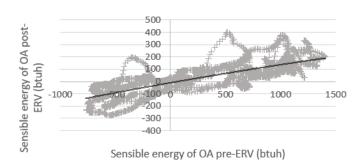


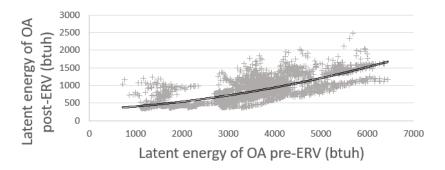


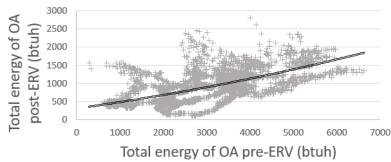


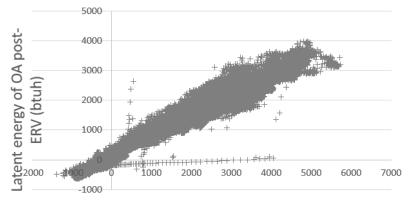


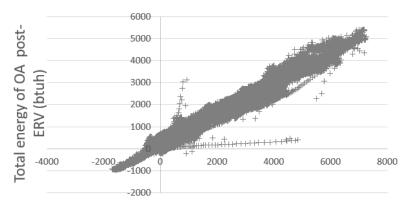






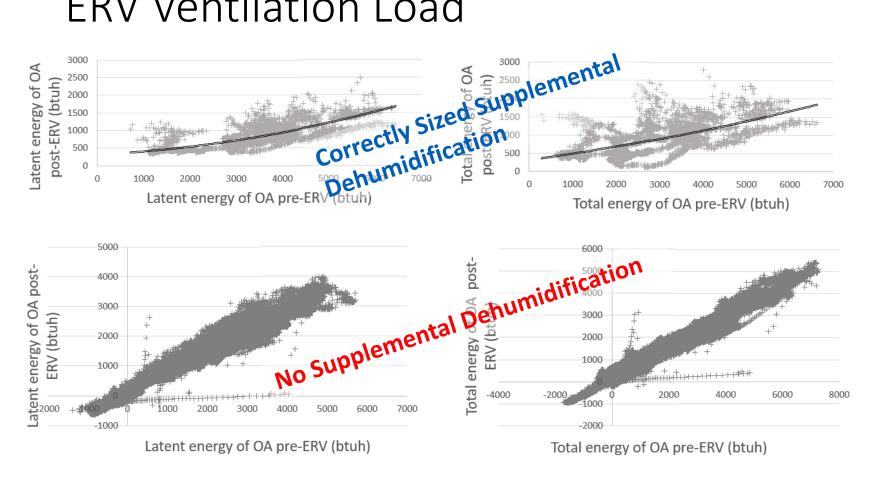




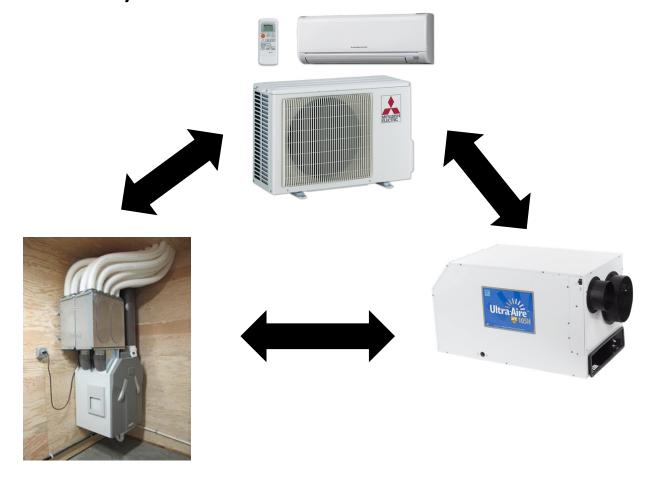


Latent energy of OA pre-ERV (btuh)

Total energy of OA pre-ERV (btuh)



Mechanical Systems



Green Grass Climate Ventilation Systems

- Typically OA is wet and near room temperature
- Need extra horsepower to deal with load from ventilation system
 - Independent of temperature
- Minimize potential for comfort issues and negative system interactions
 - Correct application
- Monitor & make adjustments



My challenge to you....

- Put Health/Safety/Durability over Energy Efficiency
- Practice true systems thinking Envelope/Mechanicals
- Determine the moisture load being brought in by ventilation system and ensure adequate latent capacity (95% of the year) – do not be afraid of dehumidification
- Educate Educate

David Treleven dtreleven@thermastor.com

